

Title: METHOD FOR MEASUREMENT OF ROTATION  
RATES/ACCELERATIONS USING A ROTATION RATE  
CORIOLIS GYRO, AS WELL AS A CORIOLIS GYRO  
WHICH IS SUITABLE FOR THIS PURPOSE

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BACKGROUND

Field of the Invention:

The present invention relates to Coriolis gyroscopes. More particularly, the invention pertains to a method for measurement of accelerations with using a rotation rate Coriolis gyro, and to a Coriolis gyro ~~which is~~ suitable for such this purpose.

Description of the Prior Art

Coriolis gyros (also referred to as "vibration gyros") are ~~being~~ increasingly employed used for navigation. ~~purposes, they have~~ Such devices include a mass system that ~~which~~ is caused to oscillate. The Each mass system generally has a large number of oscillation modes, ~~which are~~ initially independent of one another. ~~In order~~ A specific oscillation mode of the mass system is artificially excited to operate the Coriolis gyro. ~~and this~~ Such mode is referred to in the following text as the "excitation oscillation".

Coriolis forces occur that ~~which~~ draw energy from the excitation oscillation of the mass system When the Coriolis gyro is rotated and ~~thus~~ transmit a further oscillation mode of the mass system ~~which is~~ (referred to below ~~in the following text~~ as the "read

oscillation"). ~~In order~~ The read oscillation is tapped off to determine rotations of the Coriolis gyro and a corresponding read signal is investigated to determine whether any changes have occurred in the amplitude of the read oscillation which represent a measure of the rotation of the Coriolis gyro.

Coriolis gyros may comprise either ~~be in the form of both~~ an open-loop or ~~system and~~ a closed-loop system. In a closed-loop system, the amplitude of the read oscillation is continuously reset to a fixed value (preferably zero) via respective control loops, and the resetting forces ~~are~~ measured.

The mass system of the Coriolis gyro ~~which is also~~ (referred to below ~~in the following text~~ as the "resonator") may ~~in this case~~ be of ~~designed in~~ widely differing designs. ~~ways~~. For example, it is possible to use an integral mass system. Alternatively, it is possible to split the mass system into separate two oscillators ~~which are~~ coupled to one another via a spring system and capable of ~~can carry out relative~~ movements relative ~~with respect~~ to one another. High dimensional accuracies can be achieved particularly in ~~particular~~ with linear double-oscillator systems that ~~which~~ comprise a coupled system ~~composed~~ of two linear oscillators. In double-oscillator systems, the spring system that ~~which~~ couples the ~~two~~ linear oscillators to one another is, in general, designed so ~~in such a way~~

that the two linear oscillators can be caused to oscillate along a first oscillation axis, with in which ~~case~~ the second oscillator additionally oscillating can ~~additionally carry out oscillations~~ along a second oscillation axis ~~which is~~ at right angles to the first oscillation axis. In such case, the movements of the second oscillator along the second oscillation axis can ~~in this case~~ be regarded as a read oscillation while ~~those and the movements~~ of the first and second oscillators along the first oscillation axis can be regarded as an excitation oscillation.

Linear double-oscillator systems have the disadvantage that the oscillations of the two linear oscillators along the first oscillation axis can cause vibrations or reflections in the gyro frame. ~~In this case,~~ (The "gyro frame" should be understood to be a mechanical, non-oscillating structure in which the oscillators are "embedded", e.g., ~~for example~~ a non-oscillating part of a silicon wafer.) The vibrations or reflections in the gyro frame can, in turn, lead to disturbances (e.g. ~~for example~~ damping effects) to ~~the~~ oscillator movements. For example, the oscillations of the first and second linear oscillators along the first oscillation axis can ~~thus~~ be disturbed by both external vibrations and accelerations which act along the first oscillation axis. Analogously ~~to this,~~ external vibrations and accelerations acting ~~which act~~ in the direction of the second oscillation axis can

disturb the oscillations of the second linear oscillator along that this oscillation axis to corrupt the measured rotation rate which in precisely the same way as ~~with all of the~~ other disturbance influences mentioned. ~~— leads to corruption of the measured rotation rate. —~~

#### SUMMARY AND OBJECTS OF THE INVENTION

It is therefore an ~~The~~ object of ~~on which~~ the present invention ~~is based is~~ to provide ~~specify~~ a Coriolis gyro ~~by means of~~ which any disturbance of the read oscillation (i.e., that is to say of the oscillation of the second linear oscillator in the direction of the second oscillation axis) as a result of ~~the~~ disturbance influences ~~mentioned above~~ can be largely avoided.

The present invention addresses the preceding and other objects by providing, in a first aspect, in order to achieve this object, the invention provides a Coriolis gyro. as claimed in patent claim 1.

Furthermore, the invention provides a method for measurement of accelerations/rotation rates using a rotation rate Coriolis gyro as claimed in patent claim 7. Advantageous refinements and developments of the idea of the invention can be found in the dependent claims.

The Coriolis Such gyro according to the invention has a first and a second resonator, ~~which are~~ each in the

form of a coupled system comprising a first and a second linear oscillator. ~~with~~ The first resonator is being mechanically/electrostatically connected/coupled to the second resonator such that the two resonators can be caused to oscillate in antiphase with respect to one another along a common oscillation axis.

In a second aspect, ~~For this reason,~~ the invention provides a method for selective or simultaneous measurement of rotation rates and accelerations with ~~This method uses~~ a rotation rate Coriolis gyro that ~~which~~ has a first and a second resonator. The resonators ~~which~~ are ~~each~~ in the form of a coupled system comprising a first and a second linear oscillator. ~~and in which~~ Rotation rates ~~to be determined~~ are determined by tapping and evaluation of the deflections of the second oscillators. ~~The method has the following steps:~~

In such method, the two resonators are caused to carry out oscillations in antiphase with one another along a common oscillation axis. The deflections of the second oscillators are compared with one another ~~in order~~ to determine an antiphase deflection component that ~~which~~ is a measure of the rotation rate to be measured and/or ~~in order~~ to determine a common in-phase deflection component, which is a measure of the acceleration to be measured. ~~and calculation of~~ The rotation rate/acceleration to be measured is then

calculated from the in-phase deflection  
component/anti-phase deflection component.

The preceding and other features of the  
invention will become further apparent from the  
detailed description that follows. Such description is  
accompanied by a set of drawing figures. Numerals of  
the drawing figures, corresponding to those of the  
written description, point to the features of the  
invention with like numerals referring to like features  
of the invention throughout.

#### BRIEF DESCRIPTION OF THE DRAWINGS

Figure 1 is a schematic illustration of ~~shows~~  
one possible embodiment of a mass system having which  
~~comprises~~ two linear oscillators, with corresponding  
control loops, for exciting ~~which are used to excite~~  
the first oscillator.

Figure 2 is a scematic illustration of a  
~~shows one~~ possible embodiment of a mass system having  
~~which comprises~~ two linear oscillators with  
corresponding measurement and control loops for a  
rotation rate  $\Omega$  and a quadrature bias  $B_0$ , as well as  
auxiliary control loops for compensation of the  
quadrature bias  $B_0$ .

Figure 3 is a schematic illustration ~~shows an~~

~~outline sketch~~ of a mass system in accordance with an  
embodiment of ~~according to~~ the invention, which  
comprises four linear oscillators, with corresponding  
measurement and control loops for a rotation rate  $\Omega$  and  
5 a quadrature bias  $B_0$ , as well as ~~the~~ auxiliary control  
loops for compensation of the quadrature bias.

Figure 4 is a block diagram of an embodiment  
of a ~~shows one preferred embodiment of the~~ control  
system for incorporation into a mass system in  
10 accordance with that illustrated in Figure 3 above.  
~~shown in Figure 3.~~

#### DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENT

~~Accordingly, the~~ A Coriolis gyro according to  
the invention has a mass system that ~~which~~ comprises  
15 two double-oscillator systems ~~that is to say~~ (two  
resonators) or four linear oscillators. ~~The~~ Antiphase  
oscillations of the two resonators with respect to one  
another ~~in this case~~ result in the center of gravity of  
the mass system remaining stationary if the two  
20 resonators are designed appropriately. This results in  
the oscillation of the mass system producing no not  
~~being able to produce any~~ external vibrations that  
~~which in turn~~ would result in disturbances in the form  
of damping/reflections. ~~Furthermore,~~ External  
25 vibrations and accelerations in the direction of the  
common oscillation axis have no influence on the  
antiphase movement of the two resonators along the

common oscillation axis.

5       The first resonator can be coupled to the  
second resonator (e.g., for example via a spring system  
that ~~which~~ connects the first resonator to the second  
resonator). A further possibility is to couple the  
first resonator to the second resonator via an  
electrostatic field. Both types of coupling may be used  
either preclusively on their own or in combination.  
~~conjunction.~~ It is sufficient, for example, for both  
10 resonators to be formed on ~~in~~ a common substrate so  
that ~~the~~ mechanical coupling is replaced by a  
mechanical connection ~~which is itself~~ provided by the  
common substrate.

15       The configurations of the first and ~~of the~~  
second resonators ~~resonator~~ are preferably identical in  
~~terms of~~ mass and shape. ~~In this case,~~ The two  
resonators may be arranged axially symmetrically with  
respect to one another with reference ~~respect~~ to an  
axis of symmetry ~~which is~~ at right angles to the common  
20 oscillation axis. That is, ~~to say~~ the first resonator  
is mapped by the axis of symmetry onto the second  
resonator. ~~However,~~ The invention is not restricted to  
this and it is sufficient for the two resonators to  
have the same mass, but to be designed with different  
25 shapes.

As ~~already~~ mentioned, the coupled resonators



are designed ~~so in such a way~~ that both linear oscillators of a resonator can be caused to oscillate in antiphase along a first oscillation axis (excitation oscillation). ~~and~~ The second linear oscillator can additionally be caused to oscillate along a second oscillation axis (read oscillation). If the first and ~~the~~ second oscillation axes are at right angles to one another, and both ~~resonators~~ are caused to oscillate in antiphase with respect to one another along the first oscillation axis (common oscillation axis), then the second oscillators are deflected in the opposite direction during rotation of the Coriolis gyro (antiphase deflection). ~~while,~~ In contrast, during acceleration of the Coriolis gyro, the second linear oscillators are deflected in the same direction (in-phase deflection). It is thus possible to measure accelerations or rotations selectively. The acceleration is measured by evaluation of an in-phase oscillation, and ~~the~~ rotation rate is measured by evaluation of an antiphase oscillation. ~~In the following text,~~ The expressions "in-phase" and "antiphase" have the following meanings: if the coordinates in the excitation direction are denoted  $x$  and those in the read direction are denoted  $y$ , then  $x_1 = x_2$ ,  $y_1 = y_2$  for in-phase oscillation and  $x_1 = -x_2$ ,  $y_1 = -y_2$  for antiphase oscillation (in this case, the index "1" denotes the first oscillator, and the index "2" the second oscillator).

The common in-phase deflection component is determined advantageously as follows: a first quadrature bias ~~which occurs~~ within the first resonator and a second quadrature bias ~~which occurs~~ within the second resonator are determined. The first and the second quadrature biases are then added and subtracted ~~in order~~ to determine a common quadrature bias component (in-phase component) and a difference quadrature bias component (antiphase component). The common quadrature bias component is proportional to the acceleration to be measured and corresponds to the common in-phase deflection component. The difference quadrature bias component (difference) corresponds to the antiphase deflection component. ~~The~~ Rotation rate can thus be measured at the same time as ~~the~~ acceleration, via the difference quadrature bias component.

In order to assist understanding of the acceleration measurement principle described above, the physical principles of a Coriolis gyro will be briefly explained ~~once~~ again below ~~in the following description,~~ using the example of a linear double-oscillator system. In general, Coriolis gyros have a quadrature bias (i.e., ~~that is to say~~ a zero error). The quadrature bias is ~~in this case~~ composed of a plurality of quadrature bias components. One of these ~~quadrature bias~~ components arises from alignment errors of the first and second linear oscillator with respect

to one another, with such ~~these~~ alignment errors being unavoidable due to ~~because of~~ manufacturing tolerances. The alignment errors between the two oscillators produce a zero error in the measured rotation rate signal.

The Coriolis force can be represented as:

$$\vec{F} = 2m\vec{v}_x\vec{\Omega}$$

$\vec{F}$  Coriolis force

$m$  Mass of the oscillator

$\vec{v}_x$  Velocity of the oscillator

$\vec{\Omega}$  Rotation rate

[1]

If the mass that ~~which~~ reacts to the Coriolis force is equal to the oscillating mass, and if the oscillator is operated at the natural frequency  $\omega$ , then:

$$2m\vec{v}_x\vec{\Omega} = m\vec{a}_c$$

[2]

The oscillator velocity is given by:

$$\vec{v}_x = \vec{v}_{x0} \sin \omega t$$

[3]

where

$\vec{v}_{x0}$  oscillator amplitude

$\omega$  = natural frequency of the oscillator

The oscillator and Coriolis accelerations are

thus given by:

$$\vec{a}_s = \vec{v}_{s0} \omega \cos \omega t$$

$$\vec{a}_c = 2 \vec{v}_{s0} \sin \omega t \times \vec{\Omega}$$

The two acceleration vectors are thus spatially at right angles to one another and are offset through 90° with respect to one another in the time function (spatial and time orthogonality).

These two criteria can be employed ~~used in~~ order to separate the oscillator acceleration  $\vec{a}_s$  from the Coriolis acceleration  $\vec{a}_c$ . The ratio of the abovementioned acceleration amplitudes  $a_c$  and  $a_s$  is:

$$\frac{a_c}{a_s} = \frac{2\Omega}{\omega} \quad [5]$$

If the rotation rate is  $\Omega = 5^\circ/\text{h}$  and the natural frequency of the oscillator is  $f_s = 10 \text{ KHz}$ , then:

$$\frac{a_c}{a_s} = 7.7 \cdot 10^{-10} \quad [6]$$

For an accuracy of  $5^\circ/\text{h}$ , undesirable couplings of the first oscillator to the second oscillator must not exceed  $7.7 \cdot 10^{-10}$ , or must be constant. ~~at this value~~ If a mass system composed of two linear oscillators ~~is used, which are~~ coupled to one another via spring elements is employed, then the

accuracy of the spatial orthogonality between the oscillation mode and the measurement mode is limited due to ~~because of~~ the alignment error of the spring elements. The Achievable accuracy (limited by manufacturing tolerances) is  $10^{-3}$  to  $10^{-4}$ . ~~The accuracy of the~~ Time orthogonality accuracy is limited by the phase accuracy of the electronics at, for example, 10 KHz, which can likewise be complied with only to at most  $10^{-3}$  to  $10^{-4}$ . This means that the ratio of the accelerations as defined above cannot be satisfied.

Realistically, the resultant error in the measured acceleration ratio  $a_c/a_s$  is:

$$\frac{a_c}{a_s} = 10^{-6} \text{ to } 10^{-8}$$

The spatial error results in a so-called quadrature bias  $B_Q$ , which, together with the time phase error  $\Delta_\phi$ , results in a bias B:

$$B_Q = 6.5 \cdot 10^6 \text{ }^\circ/\text{h to } 6.5 \cdot 10^5 \text{ }^\circ/\text{h}$$

$$\Delta_\phi = 10^{-3} \text{ to } 10^{-4}$$

$$B = B_Q \cdot \Delta_\phi = 6,500 \text{ }^\circ/\text{h to } 65 \text{ }^\circ/\text{h}$$

[8]

The quadrature bias thus results in a major limitation ~~restriction~~ to the measurement accuracy. In this case, it should be noted that the preceding ~~above~~ error analysis takes account only of the direct coupling of the oscillation mode to the read mode. Further quadrature bias components also exist and occur, for example, as a result of couplings with other

oscillation modes.

If the Coriolis gyro is designed ~~so in such a~~  
~~way~~ that the first oscillators are connected by first  
spring elements to a gyro frame of the Coriolis gyro,  
5 and the second oscillators are connected by second  
spring elements to ~~in each case~~ one of the first  
oscillators, then the acceleration to be measured  
results in a change in the mutual alignment of the  
first oscillators with respect to the second  
10 oscillators. ~~and~~ This is particularly ~~in particular~~  
manifested in a change in ~~the~~ alignment of the second  
spring elements. The alignment change of the second  
spring elements in ~~this~~ such case produces an  
"artificial" quadrature bias component (i.e., ~~that is~~  
15 ~~to say~~ an "error") in the quadrature bias signal. It is  
thus also indirectly possible to use the determination  
of the quadrature bias to deduce the acceleration to be  
measured, which produces the corresponding "artificial"  
quadrature bias component.

20 The alignments of the first and second spring  
elements are preferably at right angles to one another.  
The spring elements may have any desired shape. The  
expression "first quadrature bias" and "second  
quadrature bias" ~~in each case~~ preferably mean the total  
25 quadrature bias of a resonator. ~~However,~~ It is also  
possible in the acceleration measurement method  
according to the invention to ~~in each case~~ determine

only one quadrature bias component in each resonator.  
In such ~~which~~ case the determined quadrature bias  
component must include at least the ~~that~~ component  
~~which is~~ produced by the acceleration to be measured or  
5 the rotation to be measured.

The Coriolis gyro preferably has a device for  
determination of first rotation rate and quadrature  
bias signals that ~~which~~ occur within the first  
resonator, and second rotation rate and quadrature bias  
10 signals that ~~which~~ occur within the second resonator.  
Furthermore, the Coriolis gyro may have a device for  
production of electrostatic fields, by means of which  
the alignment angle of the first spring elements with  
respect to the gyro frame can be varied and/or the  
15 alignment angle of the second spring elements can be  
varied with respect to the first oscillators. The  
alignment/strength of the electrostatic fields can then  
be regulated by provision of appropriate control loops  
so ~~such~~ that the first and the second quadrature bias  
20 are in each case as small as possible. A computation  
unit can use the first and second rotation  
rate/quadrature bias signals to determine the rotation  
rate, and ~~can use~~ an in-phase component of the  
electrostatic fields which compensate for the first and  
25 second quadrature biases, to deduce the acceleration to  
be measured.

The quadrature bias is thus preferably

eliminated at its point of origin. ~~itself, that is to~~  
say mechanical alignment errors of the two oscillators  
with respect to one another and changes in the mutual  
alignment of the two oscillators caused by the  
5 acceleration/rotation to be measured are compensated  
for by ~~means of~~ an electrostatic force produced by the  
electrostatic field that ~~which~~ acts on one or both  
oscillators. ~~and is~~ This type of quadrature bias  
compensation has the advantage that both rotation rates  
10 and accelerations can be determined with increased  
measurement accuracy.

In one ~~particularly~~ preferred embodiment, the  
electrical fields change the alignment angles of the  
first and second spring elements ~~in order~~ to make the  
15 alignments of the first and second spring elements  
orthogonal ~~with respect~~ to one another.  
Orthogonalization ~~such as this~~ results in compensation  
for the quadrature bias (component) produced. ~~in this~~  
way Further contributions to the quadrature bias are  
20 used to set the error angle with respect to  
orthogonality so ~~such~~ that the overall quadrature bias  
disappears. The alignment angles of the second spring  
elements with respect to the first oscillator are  
preferably varied by ~~means of~~ the electrostatic field,  
25 and the alignment angles of the first spring elements  
with respect to the gyro frame of the Coriolis gyro are  
not changed. ~~However,~~ It is also possible to use the  
electrostatic field to vary only the alignment angles



of the first spring elements or to vary the alignment angles of both the first and ~~the~~ second spring elements.

One ~~particularly~~ preferred embodiment of a Coriolis gyro according to the invention has:

- an ("overall") resonator, which is in the form of a system comprising two coupled first (linear) oscillators ("sub-resonators") ~~which are~~ excited in antiphase and each containing ~~contain~~ a second linear read oscillator,
- a device for production of at least one electrostatic field, by means of which the alignment of the two coupled first oscillators with respect to the second (read) oscillators can be varied,
- a device for determination of the quadrature biases of the read oscillators that ~~which~~ are caused by alignment errors of the two oscillators with respect to the excitation oscillator and further coupling mechanisms,
- a control loop which in each case regulates the intensity of the at least one electrostatic field by means of at least one corresponding control signal such that the determined quadrature biases are as small as possible,
- a computation unit, which in each case forms differences and sums of the at least one control signal and uses them to determine the rotation rate and the acceleration.

In principle, it is possible to calculate accelerations and rotation rates solely ~~just~~ on the basis of the determined quadrature biases. ~~that is to say~~ It is not absolutely essential to compensate for the first and second quadrature bias ~~in order~~ to determine the quadrature biases. However, this is advisable for measurement accuracy ~~reasons, since~~ as phase tolerances results in mixing the rotation rate and the quadrature ~~being mixed~~ with one another. The invention covers both alternatives.

It has also been found ~~to be~~ advantageous for each of the second oscillators to be attached to or clamped ~~in~~ on the first oscillator "at one end" in the resonators. "Clamped in at one end" can ~~in this case~~ be understood not only in the sense of the literal wording but also in a general sense. In general, attached or clamped in at one end means that the force is introduced from the first oscillator to the second oscillator essentially from one "side" of the first oscillator. If, ~~by way of~~ for example, the oscillator system were to be designed in such a way that the second oscillator were ~~is~~ bordered by the first oscillator and ~~is~~ connected to it by ~~means of~~ second spring elements, then the expression clamped in or attached at one end would imply the following: the second oscillator is readjusted for the movement by the first oscillator, by the first oscillator alternately

"pushing" or "pulling" the second oscillator by means of the second spring elements.

Clamping the second oscillator in at one end on the first oscillator has the advantage that, when an electrostatic force is exerted on the second oscillator as a result of the alignment/position change of the second oscillator ~~which results from this~~, the second spring elements can be slightly curved, thus making it possible, without any problems, to vary the corresponding alignment angle of the second spring elements. If the second oscillator ~~in this example~~ were to be attached to additional second spring elements so ~~in such a way~~ that, during movement of the first oscillator, the second oscillator were at the same time to be "pulled" and "pushed" by the second spring elements, then this would be equivalent to the second oscillator being clamped in or attached "at two ends" to the first oscillator (with the force being introduced to the second oscillator from two opposite ends of the first oscillator). In such ~~this~~ case, the additional second spring elements would produce corresponding opposing forces when an electrostatic field is applied, so that changes in the alignment angles of the second spring elements could be achieved only with difficulty. However, clamping in at two ends is acceptable when the additional second spring elements are designed so ~~such~~ that the influence of these spring elements is small so that all of the

spring elements can bend without any problems. ~~in this case as well,~~ That is, ~~to say~~ the clamping in is effectively at one end.

Depending on the design of the oscillator,  
5 ~~structure~~ clamping in at one end can be effectively provided just by the "influence" (force introduction) of the additional second spring elements being 40% or less. However, this value does not present any limitation on restriction to the invention. ~~and~~ It is  
10 also feasible for the influence of the second spring elements to be more than 40%. ~~By way of~~ For example, clamping in at one end can be achieved by all of the second spring elements that ~~which~~ connect the second oscillator to the first oscillator being arranged  
15 parallel and on the same plane. ~~as one another~~ All start and end points of the second spring elements are in each case attached to the same ends of the first and second oscillator. The start and end points of the second spring elements may ~~in this case~~ each  
20 advantageously be on a common axis, with the axes intersecting the second spring elements at right angles.

If the second oscillator is attached to or  
clamped on the first oscillator at one end, then the  
25 first spring elements are preferably designed to ~~such that they~~ clamp the first oscillator in on the gyro frame at two ends (the expressions "at one end" and "at

two ends" can be used analogously ~~here~~). As an alternative, ~~to this~~ however, it is possible for the spring elements also to be designed to in such a way ~~that they~~ clamp in the first oscillator at one end. By ~~way of~~ For example, all the first spring elements that ~~which~~ connect the first oscillator to the gyro frame of the Coriolis gyro can be arranged parallel and on the same plane as one another, with the start and end points of the first spring elements in each case preferably being located on a common axis. It is equally possible for the spring elements to be designed so in such a way that the first oscillator is clamped in on the gyro frame at one end, and the second oscillator is clamped in at two ends by the first oscillator. It is also possible for both oscillators to be clamped in at two ends. For quadrature bias compensation, it has been found to be advantageous for at least one of the two oscillators to be clamped in at one end.

~~The invention will be explained in more detail in the following text with reference to one exemplary embodiment in the figures, in which:~~

Figure 1 illustrates ~~shows~~ the schematic design of a linear double oscillator 1 with corresponding electrodes including ~~as well as~~ a block diagram of associated evaluation/excitation electronics 2. The linear double oscillator 1 is preferably

produced by ~~means of etching processes from~~ a silicon wafer. ~~and~~ It has a first linear oscillator 3, a second linear oscillator 4, first spring elements  $5_1$  to  $5_4$ , second spring elements  $6_1$  and  $6_2$  as well as parts of an intermediate frame  $7_1$  and  $7_2$  and ~~of~~ a gyro frame  $7_3$  and  $7_4$ . The second oscillator 4 is mounted within the first oscillator 3 ~~to such that it can~~ oscillate, and is connected to it via the second spring elements  $6_1$ ,  $6_2$ . The first oscillator 3 is connected to the gyro frame  $7_3$ ,  $7_4$  by ~~means of~~ the first spring elements  $5_1$  to  $5_4$  and the intermediate frame  $7_1$ ,  $7_2$ .

~~Furthermore,~~ First excitation electrodes  $8_1$  to  $8_4$ , first read electrodes  $9_1$  to  $9_4$ , second excitation electrodes  $10_1$  to  $10_4$ , and second read electrodes  $11_1$  and  $11_2$  are also provided. All of the electrodes are mechanically connected to the gyro frame, although but ~~are~~ electrically isolated. [The expression "gyro frame" refers to means a mechanical, non-oscillating structure in which the oscillators are "embedded", e.g., for ~~example~~ the non-oscillating part of the silicon wafer].

When ~~if~~ the first oscillator 3 is excited by ~~means of~~ the first excitation electrodes  $8_1$  to  $8_4$  to oscillate in the  $X_1$  direction, such ~~then this~~ movement is transmitted through the second spring elements  $6_1$ ,  $6_2$  to the second oscillator 4 (alternate "pulling" and "pushing"). The vertical alignment of the first spring elements  $5_1$  to  $5_4$  prevents the first oscillator 3 from

moving in the X2 direction. However, a vertical oscillation can be carried out by the second oscillator 4 as a result of the horizontal alignment of the second spring elements 6<sub>1</sub>, 6<sub>2</sub>. When corresponding Coriolis forces ~~accordingly~~ occur, then the second oscillator 4 is excited to oscillate in the X2 direction.

A read signal ~~that which~~ is read from the first read electrodes 9<sub>1</sub> to 9<sub>4</sub> and ~~is~~ proportional to the amplitude/frequency of the X1 movement of the first oscillator 3 is supplied, via appropriate amplifier elements 21, 22 and 23, to an analog/digital converter 24. An appropriately digitized output signal from the analog/digital converter 24 is demodulated ~~not only~~ by a first demodulator 25 ~~and but also~~ by a second demodulator 26 to form corresponding output signals, with the two demodulators operating with an offset of 90° with respect to one another. The output signal from the first demodulator 25 whose output signal controls a frequency generator 30 ~~so such~~ that the signal ~~occurring which occurs~~ downstream from the demodulator 25 is regulated at zero is supplied to a first regulator 27 ~~in order~~ to regulate the frequency of the excitation oscillation (the oscillation of the mass system 1 in the X1 direction). Analogously ~~to this~~, the output signal from the second demodulator 26 is regulated at a constant value ~~which is~~ (predetermined by the electronics component 29). A second regulator 31 ~~insures ensures~~ that the amplitude of the excitation

oscillation is regulated. The output signals from the frequency generator 30 and ~~from~~ the amplitude regulator 31 are multiplied by one another at ~~by means of~~ a multiplier 32. An output signal from the multiplier 32, ~~which is~~ proportional to the force to be applied to the first excitation electrodes  $8_1$  to  $8_4$ , acts not only on a first force/voltage converter 33 but also on a second force/voltage converter 34, which use the digital force signal to produce digital voltage signals. The digital output signals from the force/voltage converters 33, 34 are converted by ~~via a~~ first and ~~a~~ second digital/analog converters ~~converter~~ 35, 36 to corresponding analog voltage signals. ~~which are~~ Such signals are then passed to the first excitation electrodes  $8_1$  to  $8_4$ . The first ~~regulator 27~~ and the second regulators ~~regulator 27~~, 31 readjust the natural frequency of the first oscillator 3 and set the amplitude of the excitation oscillation to a specific, predeterminable value.

When Coriolis forces occur, resultant the movement of the second oscillator 4 in the X2 direction (read oscillation) ~~that results from this~~ is detected by the second read electrodes  $11_1$ ,  $11_2$ , and a read signal, ~~which is~~ proportional to the movement of the read oscillation, is supplied via appropriate amplifier elements 40, 41 and 42 to an analog/digital converter 43 (see Figure 2). A digital output signal from the analog/digital converter 43 is demodulated by a third



demodulator 44 in phase with the direct-bias signal and  
~~is~~ demodulated by a fourth demodulator 45, offset  
through  $90^\circ$ . A corresponding output signal from the  
first demodulator 44 is applied to a third regulator  
5 46, whose output signal is a compensation signal that  
~~and~~ corresponds to the rotation rate  $\Omega$  to be measured.  
An output signal from the fourth demodulator 45 is  
applied to a fourth regulator 47 whose output signal is  
a compensation signal ~~and is~~ proportional to the  
10 quadrature bias to be compensated. ~~for~~ The output  
signal from the third regulator is modulated by ~~means~~  
~~of~~ a first modulator 48, and the output signal from the  
fourth regulator 47 is modulated in an analogous manner  
~~to this by means of~~ a second modulator 49, so that  
15 amplitude-regulated signals are produced whose  
frequencies correspond to the natural frequency of the  
oscillation in the X1 direction ( $\sin \approx 0^\circ$ ,  $\cos \approx 90^\circ$ ).  
Corresponding output signals from the modulators 48, 49  
are added in an addition stage 50, whose output signal  
20 is supplied both to a third force/voltage converter 51  
and to a fourth force/voltage converter 52. The  
corresponding output signals for the force/voltage  
converters 51, 52 are supplied to digital/analog  
converters 53, 54, whose analog output signals are  
25 applied to the second excitation electrodes 10<sub>2</sub> to 10<sub>3</sub>,  
and reset the oscillation amplitudes of the second  
oscillator 4.

The electrostatic field ~~which is~~ produced by

the second excitation electrodes  $10_1$  and  $10_4$  (or the two electrostatic fields ~~which are~~ produced by the electrode pairs  $10_1, 10_3$  and  $10_2, 10_4$ ) results in an alignment/position change of the second oscillator 4 in the X2 direction, and thus in a change in the alignments of the second spring elements  $6_1$  to  $6_2$ . The fourth regulator 47 regulates the signal ~~which is~~ applied to the second excitation electrodes  $10_1$  and  $10_4$ , ~~so in such a way~~ that the quadrature bias ~~which is~~ included in the compensation signal of the fourth regulator 47 is as small as possible, or disappears. A fifth regulator 55, a fifth and a sixth force/voltage converter 56, 57 and two analog/digital converters 58, 59 are used for this purpose.

The output signal from the fourth regulator 47, which is a measure of the quadrature bias, is supplied to the fifth regulator 55, ~~that which~~ regulates the electrostatic field ~~that is~~ produced by the two excitation electrodes  $10_1$  and  $10_4$ , so that in ~~such a way that~~ the quadrature bias  $B_0$  disappears. ~~For this purpose,~~ An output signal from the fifth regulator 55 is ~~in each case~~ supplied to the fifth and sixth force/voltage converters 56, 57, for this, employing ~~which use~~ the digital force/output signal from the fifth regulator 55 to produce digital voltage signals ~~that These~~ are then converted to analog voltage signals in the analog/digital converters 58, 59. The analog output signal from the analog/digital converter 58 is

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supplied to the second excitation electrode 10<sub>1</sub> or  
(alternatively to electrode 11<sub>1</sub>). The analog output  
signal from the analog/digital converter 59 is supplied  
to the second excitation electrode 10<sub>4</sub> or (alternatively  
5 to electrode 11<sub>2</sub>).

As ~~Since~~ the second oscillator 4 is clamped  
in only by the second spring elements 6<sub>1</sub> to 6<sub>2</sub> (~~clamping~~  
clamped in at one end), such ~~the~~ alignment of the ~~these~~  
spring elements can be varied without problem by the  
10 electrostatic field ~~any problems~~. It is additionally  
~~also~~ possible to provide additional second spring  
elements, ~~which result~~ resulting in the second  
oscillator 4 being clamped in at two ends, provided  
that such ~~these~~ additional spring elements are  
15 appropriately designed to insure ~~ensure~~ that clamping  
in at one end is effective. ~~effectively achieved~~. In  
order to permit ~~allow~~ the same effect for the spring  
elements 5<sub>1</sub>, 5<sub>2</sub> (and for the spring elements 5<sub>3</sub>, 5<sub>4</sub> as  
well) the third and fourth spring elements 5<sub>3</sub>, 5<sub>4</sub>, as  
20 well as ~~and~~ the first and second spring elements 5<sub>1</sub>, 5<sub>2</sub>  
may be omitted, ~~thus~~ resulting in the first oscillator  
3 being clamped in at one end (together with an  
appropriately modified electrode configuration, ~~which~~  
~~is not shown here~~). In such a situation ~~such as this~~,  
25 the second oscillator 4 ~~could~~ may also be attached to  
the first oscillator by ~~means of~~ further spring  
elements ~~in order~~ to achieve clamping in at two ends.

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A ~~One~~ preferred embodiment of the Coriolis gyro ~~of according to~~ the invention as well as ~~and~~ its method of operation will be described in more detail ~~in the following description~~ with reference to Figure 3, a  
5 schematic illustration of a mass system comprising four linear oscillators with corresponding measurement and control loops for rotation rate and quadrature mass, as well as auxiliary control loops for compensation of the quadrature bias. ~~Figure 3 shows~~ The schematic layout of  
10 coupled system 1' comprises ~~comprising~~ a first resonator  $70_1$  and a second resonator  $70_2$ . The first resonator  $70_1$  is coupled to the second resonator  $70_2$  by  
~~via~~ a mechanical coupling element (a spring) 71. The first and the second resonator  $70_1$ ,  $70_2$  are formed in a  
15 common substrate and may ~~can~~ be caused to oscillate in antiphase with respect to one another along a common oscillation axis 72. The first and ~~the~~ second  
resonators ~~resonator~~  $70_1$ ,  $70_2$  are identical, and are mapped onto one another via an axis of symmetry 73. The  
20 design of the first and ~~of the~~ second resonator  $70_1$ ,  $70_2$  has ~~already~~ been explained in conjunction with Figures 1 and 2 and will therefore not be explained again.  
(Identical and mutually corresponding components or component groups are identified by the same reference  
25 numbers with identical components ~~which are~~ associated with different resonators being identified by different indices.)

A ~~One~~ major difference between the double

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oscillators shown in Figure 3 and those ~~the double~~  
~~oscillators shown~~ in Figures 1 and 2 is that some of  
the individual electrodes are physically combined to  
form one overall electrode. For example, the individual  
5 electrodes ~~which are~~ identified by the reference  
numbers  $8_1$ ,  $8_2$ ,  $9_1$  and  $9_2$  in Figure 3 ~~thus~~ form a common  
electrode. Further, ~~Furthermore,~~ the individual  
electrodes ~~which are~~ identified by the reference  
numbers  $8_3$ ,  $8_4$ ,  $9_3$  and  $9_4$  form a common electrode, ~~and~~  
10 those with the reference numbers  $10_4$ ,  $10_2$ ,  $11_2$  as well  
as the reference numbers  $11_1$ ,  $10_3$  and  $10_1$  each form an  
overall electrode. The same applies in an analogous  
manner to the other double-oscillator system.

During operation of the coupled system 1' in  
15 accordance with ~~according to~~ the invention, the two  
resonators  $70_1$ ,  $70_2$  oscillate in antiphase along the  
common oscillation axis 72. The coupled system 1' is  
thus not susceptible to external disturbances or to  
those ~~disturbances which are~~ emitted by the coupled  
20 system 1' itself into the substrate in which the  
resonators  $70_1$  and  $70_2$  are mounted.

When the coupled system 1' is rotated, ~~then~~  
the second oscillators  $4_1$  and  $4_2$  are deflected in  
mutually opposite directions (i.e., in the X2 direction  
25 ~~and in the opposite direction~~ to the X2 direction).  
When an acceleration of the coupled system 1' occurs,  
~~then~~ the second oscillators  $4_1$ ,  $4_2$  are each deflected in

the same direction, i.e., specifically in the same direction as the acceleration provided that such this acceleration is in the X2 direction, or in the opposite direction. ~~to it~~ Accelerations and rotations can thus  
5 be measured simultaneously or selectively. Quadrature bias compensation can be carried out ~~at the same time~~ during the measurement process in the resonators 70<sub>1</sub>, 70<sub>2</sub>. However, this is not absolutely essential.

In principle, it is possible to operate the  
10 coupled system 1' on the basis of the evaluation/excitation electronics 2 described with reference to ~~in~~ Figures 1 and 2. ~~However,~~ An alternative method (carrier frequency method) is ~~used~~ instead used ~~of this~~ in the embodiment of ~~shown in~~ Figure 3. Such ~~This~~ operating method will be described  
15 below. ~~in the following text.~~

The evaluation/excitation electronics 2 ~~which~~ ~~are~~ identified by the reference number 2' include ~~have~~ three control loops: a first control loop for  
20 excitation and/or control of an antiphase oscillation of the first oscillators 3<sub>1</sub> and 3<sub>2</sub> along the common oscillation axis 72, a second control loop for resetting and compensation of the oscillations of the second oscillator 4<sub>1</sub> along the X2 direction, and a  
25 control loop for resetting and compensation of the oscillations of the second oscillator 4<sub>2</sub> along the X2 direction. The three described control loops include

have an amplifier 60, an analog/digital converter 61, a signal separation module 62, a first to third demodulation module 63<sub>1</sub> to 63<sub>3</sub>, a control module 64, an electrode voltage calculation module 65, a carrier frequency addition module 67, and a first to sixth digital/analog converter 66<sub>1</sub> to 66<sub>6</sub>.

Carrier frequencies can be applied to the electrodes 8<sub>1</sub> to 8<sub>8</sub>, 9<sub>1</sub> to 9<sub>8</sub>, 10<sub>1</sub> to 10<sub>8</sub> and 11<sub>1</sub> to 11<sub>4</sub> for tapping excitation of the antiphase oscillation or of the oscillations of the second oscillators 4<sub>1</sub>, 4<sub>2</sub>. This may be accomplished in a number of ways. They include : a) using three different frequencies, with one frequency ~~being~~ associated with each control loop, b) using square-wave signals with a time-division multiplexing method, and or c) using random phase scrambling (stochastic modulation method).

The carrier frequencies are applied to the electrodes 8<sub>1</sub> to 8<sub>8</sub>, 9<sub>1</sub> to 9<sub>8</sub>, 10<sub>1</sub> to 10<sub>8</sub> and 11<sub>1</sub> to 11<sub>4</sub> via the associated signals UyAo, UyAu (for the second oscillator 4<sub>1</sub>), ~~and~~ Ux1, Uxr (for the antiphase resonance of the first oscillators 3<sub>1</sub> to 3<sub>2</sub>) and as well as UyBu and UyBo (for the second oscillator 4<sub>2</sub>) that ~~which~~ are produced in the carrier frequency addition module 67 and ~~are~~ excited in antiphase with respect to the abovementioned frequency signals. The oscillations of the first and second oscillators 3<sub>1</sub>, 3<sub>2</sub>, 4<sub>1</sub> and 4<sub>2</sub> are tapped off via those parts of the gyro frame ~~which are~~

identified by the reference numbers  $7_7$ ,  $7_9$ ,  $7_{11}$  and  $7_{13}$ ,  
~~and in this case are additionally~~ (used as tapping  
electrodes in addition to their function as suspension  
points for the mass system). For this ~~purpose~~, the two  
5 resonators  $70_1$ ,  $70_2$  are preferably ~~and advantageously~~  
designed to be electrically conductive, with all of the  
frames, springs and connections. The signal, ~~which is~~  
tapped off by means of the gyro frame parts  $7_7$ ,  $7_9$ ,  $7_{11}$   
and  $7_{13}$  and ~~is~~ supplied to the amplifier 60, contains  
10 information about all three oscillation modes. It ~~and~~  
is converted by the analog/digital converter 61 to a  
digital signal ~~which is~~ supplied to the signal  
separation module 62.

The assembled signal is separated into three  
15 different signals in the signal separation module 62: x  
(which contains information about the antiphase  
oscillation), yA (which contains information about the  
deflection of the second oscillator  $4_1$ ) ~~and as well as~~  
yB (which contains information about the deflection of  
20 the second oscillator  $4_2$ ). The signals are separated  
differently in accordance with ~~depending on~~ the type of  
carrier frequency method ~~used~~ (see a) to c) above). ~~and~~  
Separation is carried out by demodulation with the  
corresponding signals of the carrier frequency method  
25 ~~that is~~ used. The signals x, yA and yB are supplied to  
the demodulation modules  $63_1$  to  $63_3$ , that ~~which~~  
demodulate them with ~~using~~ an operating frequency of  
the antiphase oscillation for  $0^\circ$  and  $90^\circ$ . The control



module 64 and ~~as well as~~ the electrode voltage calculation module 65 for regulation/calculation of the signals  $F_{x1/r}$  or  $U_{x1/r}$ , respectively, are preferably configured analogously to the electronics module 2 of ~~shown in~~ Figure 1. The control module 64 and the electrode voltage calculation module 65 (for regulation/calculation of the signals  $F_{yAo/u}$ ,  $U_{yAo/u}$ , and  $F_{yBo/u}$ ,  $U_{yBo/u}$ ) are preferably designed analogously to the electronics module 2 of ~~shown in~~ Figure 2.

Figure 4 is a block diagram of an embodiment of a control system for incorporation into a mass system in accordance with Figure 3. It shows one preferred embodiment of the control system ~~that is~~ identified by the reference number 64 in Figure 3. The control system 64 includes ~~has~~ a first to third part 64<sub>1</sub> to 64<sub>3</sub>. The first part 64<sub>1</sub> has a first regulator 80, a frequency generator 81, a second regulator 82, an electronics component 83, an addition stage 84 and a multiplier 85. The ~~method of~~ operation of the first part corresponds essentially to that ~~the method of~~ ~~operation~~ of the electronics module 2 of ~~shown in~~ Figure 1 and will therefore not be described once again ~~here~~. The second part 64<sub>2</sub> has a first regulator 90, a first modulator 91, a second regulator 92, a second modulator 93 and a third regulator 94. A first and a second addition stage 95, 96 are also provided. A rotation rate signal  $\Omega$  can be determined at the output of the first regulator 90, and an assembled signal

comprising a quadrature bias  $B_0$  and an acceleration  $A$  can be determined at the output of the third regulator 94.

5 The third part 64<sub>3</sub> of the control system 64 has a first regulator 100, a first modulator 101, a second regulator 102, a second modulator 103 and a third regulator 104. A first and a second addition stage 105, 106 are also provided. A rotation rate signal  $\Omega$  with a negative mathematical sign can be  
10 tapped off at the output of the first regulator 100 and an assembled signal comprising the quadrature bias  $B_0$  with a negative mathematical sign and an acceleration signal  $A$  can be tapped off at the output of the third regulator 104. The method of operation of the second and of the third parts ~~part~~ 64<sub>2</sub> and 64<sub>3</sub> corresponds to  
15 that of the electronics module 2 illustrated in Figure 2, and will therefore not be explained ~~once~~ again ~~here~~.

Only the signals for resetting ~~of the~~ rotation rate and ~~the~~ quadrature, after ~~the~~ multiplication by the operating frequency, are passed,  
20 together with the DC voltages for the quadrature auxiliary regulator, to a combined electrode pair. The two signals are therefore added so that the calculation of the electrode voltages includes the reset signals  
25 for ~~the~~ oscillation frequency and the DC signal for quadrature regulation. The electrode voltages  $U_{x1/r}$ ,  $U_{yA0/u}$  and  $U_{yB0/u}$  thusly ~~calculated in this way~~ are

then added to the carrier frequency signals and ~~are~~  
jointly passed via the analog/digital converters 66<sub>1</sub> to  
66<sub>6</sub> to the electrodes.

5 The carrier frequency methods described above  
with antiphase excitation have the advantage that a  
signal is applied to the amplifier 60 only when the  
linear oscillators 3<sub>1</sub>, 3<sub>2</sub>, as well as 4<sub>1</sub> and 4<sub>2</sub>, are  
deflected. The frequency signals ~~which are~~ used for  
excitation may be discrete frequencies or square-wave  
10 signals. Square-wave excitation is preferred, as it is  
easier to produce and process.

A number of analyses relating to the  
measurement accuracy of the acceleration measurement  
method according to the invention are ~~will also be~~  
15 described in the following text.

The Rotation rate results in an antiphase  
deflection of the oscillators 4<sub>1</sub> and 4<sub>2</sub> at the operating  
frequency of the Coriolis gyro. In contrast,  
acceleration results in an in-phase deflection of the  
20 oscillators 4<sub>1</sub> and 4<sub>2</sub> with ~~in which case~~ the  
acceleration ~~can be~~ measured in the frequency range  
from 0 Hz to about 500 Hz with a measurement accuracy  
of 50 mg to 50 ig.

25 The in-phase deflection to be measured is  
given by:

$$\alpha = \frac{a}{\ell \cdot \omega^2}$$

$\alpha$  Deflection angle

$a$  Acceleration

$\ell$  Length of the spring

$\omega$  Natural frequency of the oscillators  $\omega_1$  to  $\omega_2$ .

For typical natural frequencies  $\omega = 2 * \pi f = 6000$  rad/s to 60000 rad/s and spring lengths of  $\ell = 1$  mm of Coriolis gyros, the measurement accuracy of, for example 5 mg is:

$$\alpha = 1.4 * 10^{-6} \text{ to } 1.4 * 10^{-8} \text{ rad or } x_2 = x_1 = 1.4 \text{ nm to } 14 \text{ pm.}$$

Small deflections such as the above ~~these~~ are difficult to measure in the frequency range from 0 to 500 Hz. At a minimum ~~the least~~, ~~this requires~~ additional electronic complexity is required for the multisensor according to the invention as ~~because~~ the electronics have to measure very accurately in both the operating range of the gyro function (rotation rate measurement) from 1 to 10 KHz and in the operating range for measurement of the acceleration from 0 to 500 Hz.

This disadvantage can be overcome in ~~according to~~ the invention by using the quadrature regulation, as described above, for a mass system comprising two linear oscillators (Figures 1 and 2) for the mass system composed of four linear oscillators

(Figure 3): the acceleration detunes the orthogonality error, ~~thus~~ resulting in an in-phase quadrature signal, ~~which can~~ clearly be seen, at the operating frequency in the oscillators  $4_1$  and  $4_2$ :

5 
$$\Omega_Q = \frac{a_Q}{a_s} \cdot \frac{\omega}{2} = \alpha \frac{\omega}{2}$$

In this case,  $\Omega_Q$  is the quadrature rotation rate,  $a_Q$  is the quadrature acceleration and  $a_s$  is the oscillator acceleration.

10 For a measurement accuracy of, for example 5 mg ( $\alpha = 1.4 \cdot 10^{-6}$  rad), this results in:

$$\Omega_Q = 0.0042 \frac{\text{rad}}{\text{s}} = 0.25^\circ/\text{s} = 866^\circ/\text{h} \quad \text{at a natural}$$

frequency of 1 kHz

$$\Omega_Q = 4.2 \cdot 10^{-5} \frac{\text{rad}}{\text{s}} / 0.0025^\circ/\text{s} = 8.7^\circ/\text{h} \quad \text{at a natural}$$

frequency of 10 kHz

15 For a rotation rate sensor of  $5^\circ/\text{h}$ , the quadrature rotation rate of  $866^\circ/\text{h}$  can be verified with certainty using the same electronics. ~~while,~~ In contrast, at the natural frequency of 10 KHz and with the quadrature rotation rate of  $8.7^\circ/\text{h}$ , the verification limit of the rotation rate sensor of  $5^\circ/\text{h}$  is virtually exhausted. Although this measurement is also stable in the long run term, it depends on the long-term stability of the quadrature rotation rate. 20 The actual quadrature rotation rate is an antiphase

signal. The stability of the acceleration measurement therefore depends on the difference in the quadrature rotation rates from the oscillator  $4_1$  to the oscillator  $4_2$ , and their stability. Since the two oscillators are located close to one another and were manufactured in one process step, it is predicted that it is possible to cover a range with low accuracy from 50 mg to 50  $\mu$ g.

While this invention has been described with reference to its presently preferred embodiment, it is not limited thereto. Rather, the invention is limited only insofar as it is defined by the following set of patent claims and includes within its scope all equivalents thereof.

## Patent Claims

### What is claimed is:

1     1.    A Coriolis gyro (1'), having a first and a second  
2     resonator (70<sub>1</sub>, 70<sub>2</sub>), which are each in the form of a  
3     coupled system comprising a first and a second linear  
4     oscillator (3<sub>1</sub>, 3<sub>2</sub>, 4<sub>1</sub>, 4<sub>2</sub>), with the first resonator  
5     (70<sub>1</sub>) being mechanically/electrostatically  
6     connected/coupled to the second resonator (70<sub>2</sub>) such  
7     that the two resonators can be caused to oscillate in  
8     antiphase with one another along a common oscillation  
9     axis (72).

1     2.    The Coriolis gyro (1') as claimed in claim 1,  
2     characterized in that the configurations of the first  
3     and of the second resonator (70<sub>1</sub>, 70<sub>2</sub>) are identical,  
4     with the resonators (70<sub>1</sub>, 70<sub>2</sub>) being arranged axially  
5     symmetrically with respect to one another with respect  
6     to an axis of symmetry (73) which is at right angles to  
7     the common oscillation axis (72).

1     3.    The Coriolis gyro (1') as claimed in claim 1 or 2,  
2     characterized in that the first oscillators (3<sub>1</sub>, 3<sub>2</sub>) are  
3     each connected by means of first spring elements  
4     (5<sub>1</sub> - 5<sub>8</sub>) to a gyro frame (7<sub>1</sub> - 7<sub>14</sub>) of the Coriolis gyro,  
5     and the second oscillators (4<sub>1</sub>, 4<sub>2</sub>) are each connected  
6     by second spring elements (6<sub>1</sub> - 6<sub>4</sub>) to one of the first  
7     oscillators (3<sub>1</sub>, 3<sub>2</sub>).

1 4. The Coriolis gyro (1') as claimed in claim 3,  
2 characterized in that the second oscillators ( $4_1$ ,  $4_2$ )  
3 are attached/clamped in at one end to the first  
4 oscillators ( $3_1$ ,  $3_2$ ) by means of the second spring  
5 elements ( $6_1 - 6_4$ ) and/or the first oscillators ( $3_1$ ,  $3_2$ )  
6 are attached/clamped in at one end to a gyro frame of  
7 the Coriolis gyro by means of the first spring elements  
8 ( $5_1 - 5_8$ ).

1 5. The Coriolis gyro (1') as claimed in claim 3 or 4,  
2 characterized by a device for production of  
3 electrostatic fields, by means of which the alignment  
4 angle of the first spring elements ( $5_1 - 5_8$ ) with  
5 respect to the gyro frame can be varied, and/or the  
6 alignment angle of the second spring elements ( $6_1 - 6_4$ )  
7 with respect to the first oscillators ( $3_1$ ,  $3_2$ ) can be  
8 varied.



1        6.    The Coriolis gyro (1') as claimed in claim 5,  
2        characterized by  
3        -       a device (10<sub>1</sub> - 10<sub>8</sub>, 11<sub>1</sub> - 11<sub>4</sub>) by means of which it  
4        is possible to determine first signals for the rotation  
5        rate and quadrature bias, which occur within the first  
6        resonator (70<sub>1</sub>), and second signals for the rotation  
7        rate and quadrature bias, which occur in the second  
8        resonator (70<sub>2</sub>),  
9        -       control loops (60 - 67) by means of which the  
10       alignment/strength of the electrostatic fields is  
11       regulated such that the first and the second quadrature  
12       bias are each as small as possible, and  
13       -       a computation unit, which uses the first and  
14       second signals to determine the rotation rate, and uses  
15       an in-phase component of the electrostatic fields which  
16       compensate for the first and second quadrature biases  
17       to determine the acceleration to be measured.

1        7.    A method for selective or simultaneous measurement  
2        of rotation rates and accelerations using a rotation  
3        rate Coriolis gyro (1') which has a first and a second  
4        resonator ( $70_1$ ,  $70_2$ ) which are each in the form of a  
5        coupled system comprising a first and a second linear  
6        oscillator ( $3_1$ ,  $3_2$ ,  $4_1$ ,  $4_2$ ), with the rotation rates  
7        being determined by tapping and evaluation of the  
8        deflections of the second oscillators ( $4_1$ ,  $4_2$ ), having  
9        the following steps:

10       -     the two resonators ( $70_1$ ,  $70_2$ ) are caused to carry  
11       out oscillations in antiphase with one another along a  
12       common oscillation axis ( $72$ ),

13       -     the deflections of the second oscillators ( $4_1$ ,  $4_2$ )  
14       are compared with one another in order to determine an  
15       antiphase deflection component which is a measure of  
16       the rotation rate to be measured and/or in order to  
17       determine a common in-phase deflection component, which  
18       is a measure of the acceleration to be measured,

19       -     calculation of the rotation rate/acceleration to  
20       be measured from the in-phase deflection  
21       component/antiphase deflection component.

1        8.    The method as claimed in claim 7,  
2        characterized in that the common in-phase deflection  
3        component is determined as follows:  
4        -       a first quadrature bias is determined which occurs  
5        within the first resonator (70<sub>1</sub>),  
6        -       a second quadrature bias is determined which  
7        occurs within the second resonator (70<sub>2</sub>),  
8        -       the first quadrature bias is calculated using the  
9        second quadrature bias in order to determine a common  
10       quadrature bias component which is proportional to the  
11       acceleration to be measured and represents the common  
12       in-phase deflection component.

1        9.    The method as claimed in claim 8,  
2        characterized in that electrostatic fields are produced  
3        in order to vary the mutual alignment of the first and  
4        second oscillators (3<sub>1</sub>, 3<sub>2</sub>, 4<sub>1</sub>, 4<sub>2</sub>), with the  
5        alignment/strength of the electrostatic fields being  
6        regulated such that the first and the second quadrature  
7        bias are each as small as possible.

~~Method for measurement of rotation rates/accelerations  
using a rotation rate Coriolis gyro, as well as a  
Coriolis gyro which is suitable for this purpose~~

#### ABSTRACT

A Coriolis gyro includes ~~(1')~~ has a first and a second resonator, ~~(70<sub>1</sub>, 70<sub>2</sub>)~~, which are each in the form of a coupled system comprising a first and a second linear oscillators. ~~(3<sub>1</sub>, 3<sub>2</sub>, 4<sub>1</sub> and 4<sub>2</sub>)~~, in which case The first resonator ~~(70<sub>1</sub>)~~ together with ~~(70<sub>2</sub>)~~ can be caused to oscillate in antiphase with respect to ~~one another~~ the second resonator along a common oscillation axis. ~~(72)~~ A system which is coupled in this way has the advantage that it is possible to measure the rotation rate and acceleration ~~accelerations~~ simultaneously, with insensitivity to disturbances and ~~that it is insensitive to disturbance influences, for example (e.g., externally or internally acting vibrations).~~

~~(Figure 3)~~